



COUPLERS OF EMU/DEMU

Role of Couplers

To Provide

Flexibility to Individual Rolling Stock **Transmit the Pulling Force** Provide Buffing at the time of Braking Requirement To withstand The Buffing Force and Pulling Force To facilitate Side/Vertical Displacement

To Coach Body at

Curve & Elevation Change

COUPLERS OF EMU/DEMU

Schaku Couplers

Original Scharfenburg Kupplung, GmbH Designed in 1903 in Germany Most Widely used in Mass Transit System around the world Semi Permanent Coupler End A End B

SCHAKU COUPLERS FEATURE

Tensile Load 70T Max. Compressive Load 100 T Max. **Rigid Connection with Negligible Play** No Appreciable Wear & Tear during Train Operation No Slack means No Jerk **Rubber Steel Composite Draft Gear** Capacity 800 Kg-M

SCHAKU COUPLERS FEATURE

Fulcrum of Coupler Located Far Allows Side Displacement of **Coaches & Couplers** At Curves **Reducing the Lateral Force On Wheels** Less Wear

Less Prone to Derailment

SCHAKU COUPLERS FEATURE

Pivot Joint between Coach and Coupler permits Horizontal Deflection 284mm/13° either side Vertical Deflection of <u>+75 mm</u> Achieved by Articulation Bearing Interlocking in Vertical Direction **Prevents Climbing of** One Coach over Other In Case of Accident

Semi Permanent Coupler Consist of

End A&B Bearing Bracket with Support -Draw and Buffer Gear -End A&B Intermediate Tube -End A&B Air Pipe Coupling -End A&B **Center Adjustment Device -**End A&B **Centering Device -**End A Only **Centering Device -**End B Only Adjustable Cup sleeve -End A only

Bearing Bracket with Support - End A&B

Fixed into End Part of Coach Houses Two Concentric Bushes Connected to Articulation Bearing Leaf Spring Holding Device Leaf Springs to support Overhanging Coupler



Draw and Buffer Gear - End A&B

Rubber Spring and Steel Plate Always in Compression Fabricated Yoke housing **Rubber Spring** Plates **Articulation Bearing** Connected to **Bearing Bracket and** Intermediate Tube



Center Adjustment Device - End A&B

Two Concentric Helical Compression Springs in Barrel with Clevis End Facilitate Mounting into Underside of Frame Exert Storing Force to keep Coupler End in Central Position



End A&B

Intermediate Tube -

Two Forged Ring welded on Seamless Tube Single Solid Stock extending from Articulation Bearing

Air Pipe Coupling - End A&B

Fabricated Structure houses Mounting for MR Pipe and Brake Pipe







Adjustable Cup Sleeve -

End A only

Mounted on Outer Part of Intermediate Tube End A Two-semi Circular Sleeve with Tapered Slot Sleeves Adjustment in Position by Spindle Forming Rigid Connection Exact Centering done by Guide Pin





Centering Device - End B Only

Circular Disc Type Forging with Guide Pins Forms Coupling Face of Intermediate Tube End A&B



MAINTENANCE - COUPLERS

Polyamide Bush **Replacement at POH Draft Gear** Height in Unrestricted Condition Intermediate Tube **DPT of Welded Joints** Cup Sleeve Wear Check by Gauge **Bearing Bracket** Wear Check by Gauge **Check for Cracks**



BOGIE OF EMU/DEMU

All Welded Light Weight Bogie Axles with Self Aligned Spherical Roller Bearing Guided Rigidly by

Telescopic Axle Guide Dash Pot Arrangement Helical Spring as Primary Suspension Supported on Side Bearers DPC – 1200 mm apart TC – 1600 mm apart

BOGIE OF EMU/DEMU

Floating Bolster resting on Pair of Air Spring as Secondary Suspension **Rest on Lower Spring Beam** Vertical and Lateral Shock Absorbers To damp Vibrations Lateral/Vertical **Centre Pivot** Facilitate Rotation of Bogie **Transmit Tractive/Braking Force Securing of Floating Bolster** in TC by Anchor Link in DPC – Between Bogie Transom

Bogie Frame DPC - DMU/DPC5-0-0-501 TC - DMU/TC4-0-0-401 Axle Guide Brackets Welded Lower Spring Beam Welded Joint & Corrosion **Rubbing Plate in DPC** Provided between Bolster and Bogie Serve as Pad Nylon or Manganese Steel **Clearance** to be maintained

Wheel Diameter

	New	Condemning	Last Shop Issue
DPC	952 mm	877 mm	885 mm
TC	915 mm	813 mm	837 mm

Permissible Variation in Wheel Tread Dia Same as ICF Coach

Axle Bearing

Self Aligning Spherical Roller Bearing DPC - EMU/M-0-2-001 TC - WTAC3-0-2- 304

Centre Pivot

Bent Wear Crack Chalk Test/DPT Loose/Missing Mounting Bolts Side Bearers Hard Wearing Ground Steel Plate Bronze Wearing Piece

Oil Bath

Dust Seal Cover Replenishment of Oil

Traction Motor Removal Electrical Manual No. MM/DC EMU.003 Shock Absorber Vertical – DMU/DPCS-0-5-503 Lateral - DC/EMUM/ASR-0-5-005 Testing **Brake Gear** Axle Guide Guide Bush **Bogie Bolster Centre Pivot Silent Block and Sleeve**

Coupler Height Adjustment Buffer Height of DPC – 1105 mm +0/-15 Schaku Coupler Height - 1035 mm +0/-15

Air Brake Brake Cylinder Piston Stroke DPC – 40 mm TC – 32 mm Brake Cylinder Pressure DPC – 1.6 kg/cm² POH Periodicity – 18 Months



AIR SPRING

Air Spring A Column of Confined Air in A Container Designed to Utilize the Compressed Air as A Medium of Transmitting Force. Ability of Air Spring to Support a Mass Depends on its Effective Area Pressure

Stiffness of Air Spring Depends on Change in Effective Area Change in Air Pressure

TYPES OF AIR SPRINGS

Locked-In System Springs Having a Constant Mass of Air Not Connected with Any Air Source The Natural Frequency Increases with Load

Active-Air System Springs Connected to a Constant Source of Air Through Height Control Valve Arrangement Fairly Uniform Natural Frequency

COMPRESSION PROCESSES

Isothermal Process A Constant Temperature Process Possible when Compression takes place in infinitesimally small steps Minimum Stiffness most desirable but practically not possible

COMPRESSION PROCESSES

Adiabatic Process

Occurs during very rapid deflection Maximum Stiffness

Polytropic Process The Most Practical One Lies between the Previous Two

MECHANICAL MODEL

Pressure of Spring Po =Pat+Pabs Force on spring F=(Po-Pat)Aeff=Pabs*Aeff Differentiating with respect to deflection dF/dz = Aeff*dPo/dz + Pabs*dAeff/dz(1)

For Polytropic Process PV**n = Constant (V**n)dPo/dz + Po*n*V**(n-1)dV/dz = 0...(2) $dPo/dz = - {Po*n*V**(n-1)/v**n}dV/dz$ = Po*n*Aeff/V

MECHANICAL MODEL

Substituting dPo/dz in (1) Stiffness K =

(Po*n*Aeff**2)/V + Pabs*dAeff/dz n is polytropic index and z is the deflection

This can be approximated as K =K1 + K2 K1 is stiffness due to change in pressure K2 is stiffness due to change in effective area

AIR SPRING ON IR

Active System an Enclosed Pressurized Air in Predefined Chamber of Rubber Bellow with Emergency Rubber Spring Air Bellow fitted Between Two Plates Air Pressure Creates Gap between Plates Provides Cushioning



AIR SPRING ON IR

Emergency Rubber Spring Comes in Operation upon Deflation of Bellow During Overload Requires Compressed Air 7 kg/cm² Able to support Maximum Load with Air Pressure inside Bellow 5.5-5.8 kg/cm²

Failure/Deflation/Heavy Leakage of Air Spring Train can move at 50 kmph

CONSTRUCTION

1.1 Airspring assembly description

Airspring system is composed of Top plate, air bellow, emergency spring and sliding plate and fasteners and O-rings. For details please see Fig1 and Table1.



Table 1: Part list for ck509 airspring

Part	Parts name	Part number	Weight (Kg)	Quantity per unit
1	O-ring	GB3452.1 75×5.3	0.1	2
2	Air inlet	C.KH060400301	5	1
3	Hexagon socket head cap screws	GB/T 5786-2000 M8×16	0.2	4
4	O-ring	GB3452.1 87.5×3.55	0.05	1
5	Top plate	C.KH060400300	42.5	1
6	Sliding plate	C.KH060400500	1.25	1
7	Hexagon socket head cap screws	GB/T 70.3 M8×20	0.3	6
8	Emergency spring	C.KH060400200	90.5	1
9	Air spring bellow	C.KH060400100	11.25	1

AIR SPRINGS – WORKING PRINCIPLE

The Fundamental Principle

Mass of Compressed Air in **Predefined Rubber Bellow** Exerts a Force (Kinetic Theory of Gases) **Functions as Suspension Element** Dynamic Forces are taken care by **Changes in Internal Pressure Air Spring Effective Area According To Height Variation**

AIR SPRINGS – WORKING PRINCIPLE The Fundamental Principle (Contd...)

When Load Added or Removed Leveling Valve Operates to Add or Remove Sufficient Air To Maintain Set Height of Air Spring The Overall Vertical and Lateral Characteristics Depends Upon The Pressure and Volume of Air Shape and Size of Air Bellows Properties of Air Used i.e. Polytropic Index.

AIR SPRINGS – WORKING PRINCIPLE

The Fundamental Principle (Contd...)





Fig. Working principle of Air spring

AIR SPRINGS – WORKING PRINCIPLE The Fundamental Principle (Contd...)



Vertical Spring Action



Lateral Spring Action

FIG. No.3 VERTICAL AND LATERAL ACTION OF AIR SPRING

AIR SPRING SYSTEM – DIAGRAM



AIR SPRING SYSTEM – COMPONENTS

Main equipments of Air suspension system: Air spring - 04 Nos /Coach 1. Levelling valve - 04 Nos /Coach 2. Duplex check valve - 02 Nos /Coach 3. 40 ltrs auxiliary reservoir - 04 Nos /Coach 4. 5. Bogie suspension isolating cock - 02 Nos /Coach Non return valve 6. - 01 No /Coach 7. 150 ltrs MR reservoir - 01 No /Coach Coach suspension isolating cock - 01 No /Coach 8.

AIR SPRINGS SYSTEM – PARTS



AIR SPRINGS – RUBBER BELLOW



AIR SPRINGS – RESERVOIR



AIR SPRINGS – DUPLEX CHECK VALVE



AIR SPRINGS – DUPLEX CHECK VALVE

Combination of Two Check Valves in Opposite Direction

Installed between Air Springs of Same Bogie

To Prevent Severe Tilt if Differential Pressure of Adjacent Bellows > 1.5 Bar

AIR SPRINGS – DUPLEX CHECK VALVE



AIR SPRINGS – INSTALLATION LEVER HORIZONTAL LEVER

Installed between Cradle and Leveling Valve Installation Lever – Setting of Length Horizontal Lever – Feedback to Leveling Valve Lever Length–305 mm Spring Height–255 mm





AIR SPRINGS – LEVELING VALVE



AIR SPRINGS – LEVELING VALVE



AIR SPRINGS – LEVELING VALVE



AIR SPRING VS STEEL SPRINGS



IN SECONDARY SUSPENSION

Benefits of Air Springs

- Load Proportionate Stiffness –
- Maintains Constant Height at Varying Loads
- Reduction in No. of Variants
- Less Failures as compared to Helical Springs Superior Ride Quality

Improvement in Ride Index(RI)

IN SECONDARY SUSPENSION

Disadvantages of Air Spring Higher Initial Cost Additional Air Pipeline required Air Pressure must be maintained Special Care to be taken in Maintenance to avoid Oil and Paint Spillage on Rubber Bellow

MAINTENANCE

Leakage Test of Air Spring At 6/9 kg/cm² (CAMTECH/RDSO 2008) Drop in Pressure within 0.2/0.1/1% in 1 hour **Installation Lever Adjustment** Height of Air Spring **Pressure in Air Spring Pressure Difference between Air Springs** For same Height Should not be more than 1.5 bar

In Case of Failure of Air Spring Failure of Supply of Air **Rupture of Air Spring Bellow** Warrants reduction in Speed In order to communicate the Crew To acknowledge Failure of Air Spring and to move at reduced speed **FIBA** is installed

Features of FIBA

02 Set per Coach – 01 set per Bogie Each set consists 02 valve per bogie To sense Pressure of each Air Spring Pressure of Any Spring goes < 1.0 kg/cm² **FIBA** actuates **Connects BP to Atmosphere Application of Full Service Brake**

Features of FIBA (Contd...)

Hissing Steady Sound of 90 + 5 dB To attract attention of Crew 04 Indicators per coach Change color Green to Red **Till Rectification of Bellow Resetting of FIBA Closing Isolating Cock** Manual Key

FIBA is to be fitted in All 1600 HP DEMU Fitted with Air Suspension (Letter 13-04-17) New Coaches by ICF Old Coaches in Zonal Railway Supply Order by ICF

RELEVANT TECHNICAL INSTRUCTIONS

RDSO Maintenance Manual No. CMI-9802 Rev.2 RDSO Technical Instruction No. CMI-K401 CAMTECH Publication CAMTECH/M/C/2010-11/Air Sus. System/1.0 For Air Suspension System Maintenance Manual for 1400 HP DEMU